

Diluted Combustion of Low Calorific, Alternative Fuels on a 30 kW furnace

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Diluted Combustion

Why



- High combustion efficiencies
- **Low NO_x & CO emissions**
- **No flame front formation**

How

- High recirculation rates
- Flue gases temperature higher than **self-ignition** fuel one

Framework

Increasing request from industry for new combustion techniques which can be safely used to burn **biogas**, **gasified waste** or **by-product gases**.

Diluted combustion can be a very interesting solution. **No flame front** is present, thus fuel with **highly variable calorific values** can be burned with no or after simple injection system modifications.

Research at UMONS

Main Objectives

- Creation of an **experimental database** for a furnace working in diluted combustion with different **alternative fuels**. Successfully tested

	N ₂	CO ₂	CH ₄	H ₂	CO	LHV (kJ/Nm ³)
CH₄			100%			35806
COG	2%	1,5%	28,5%	62%	6%	17652
B50	28%	12%	14,25%	32,5%	13,25%	10282
Biogas		40%	60%			21484
Syngas	45,5%	12%	1,5%	20%	21%	5346

- Understanding of **main phenomena** and the influence of **changes in main parameters setting** for the different fuels. Main investigated effects:

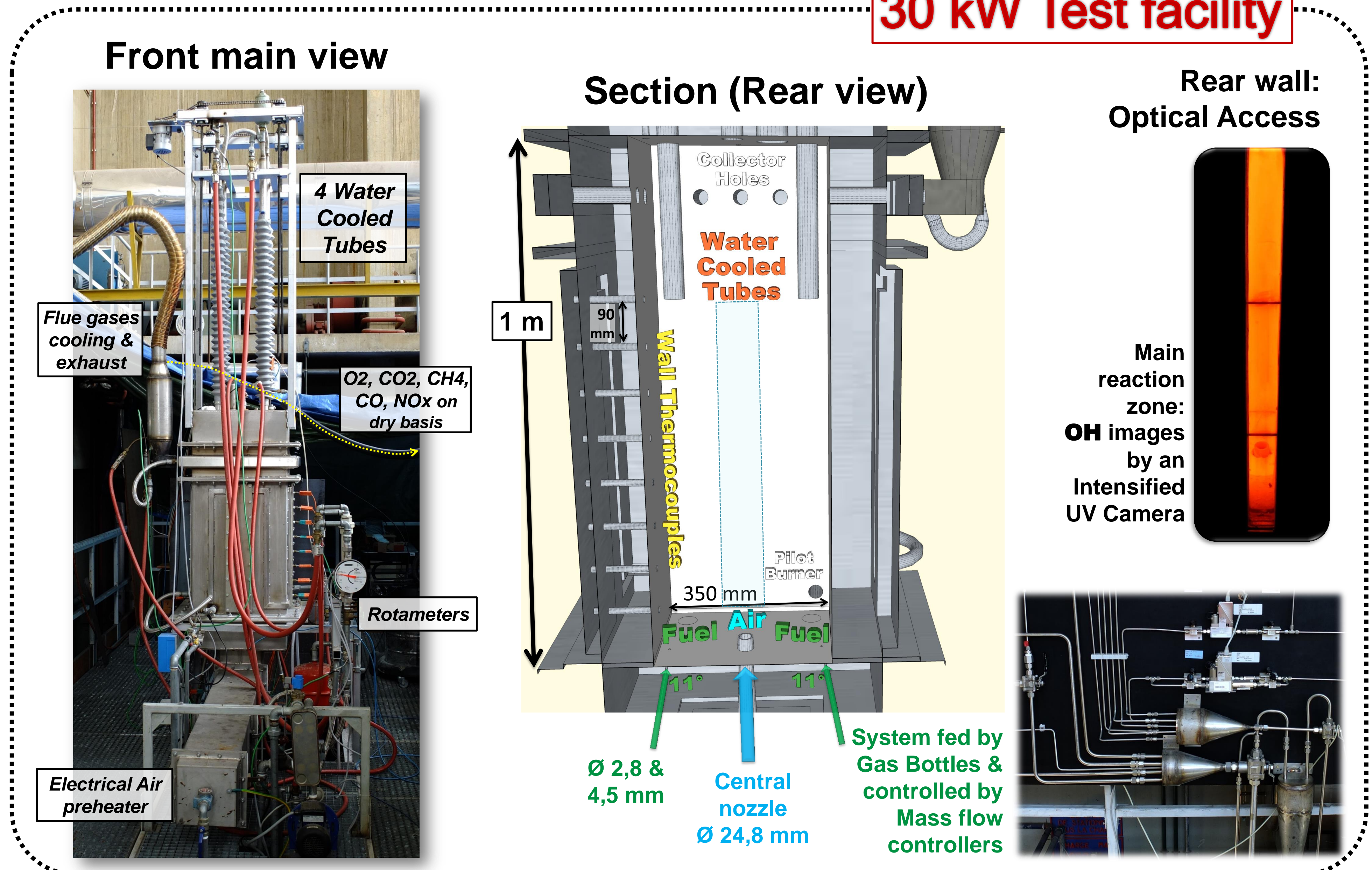
- *Air preheating temperature*
- *Immersion of the water cooled tubes*
- *Diameter of the fuel injectors*
- *Reduced fuel power*

Test campaign 2014

Biogas
CH₄
Syngas*

30kW Fuel power (*15kW)
Excess Air 15%
Air preheating 800°C
Fuel injectors Ø 2,8 & 4,5* mm
Window: Opened* (& Closed for Biogas)
Immersion 10* – 20 – 30 cm

30 kW Test facility



Test Campaign 2014: Results with Opened Window

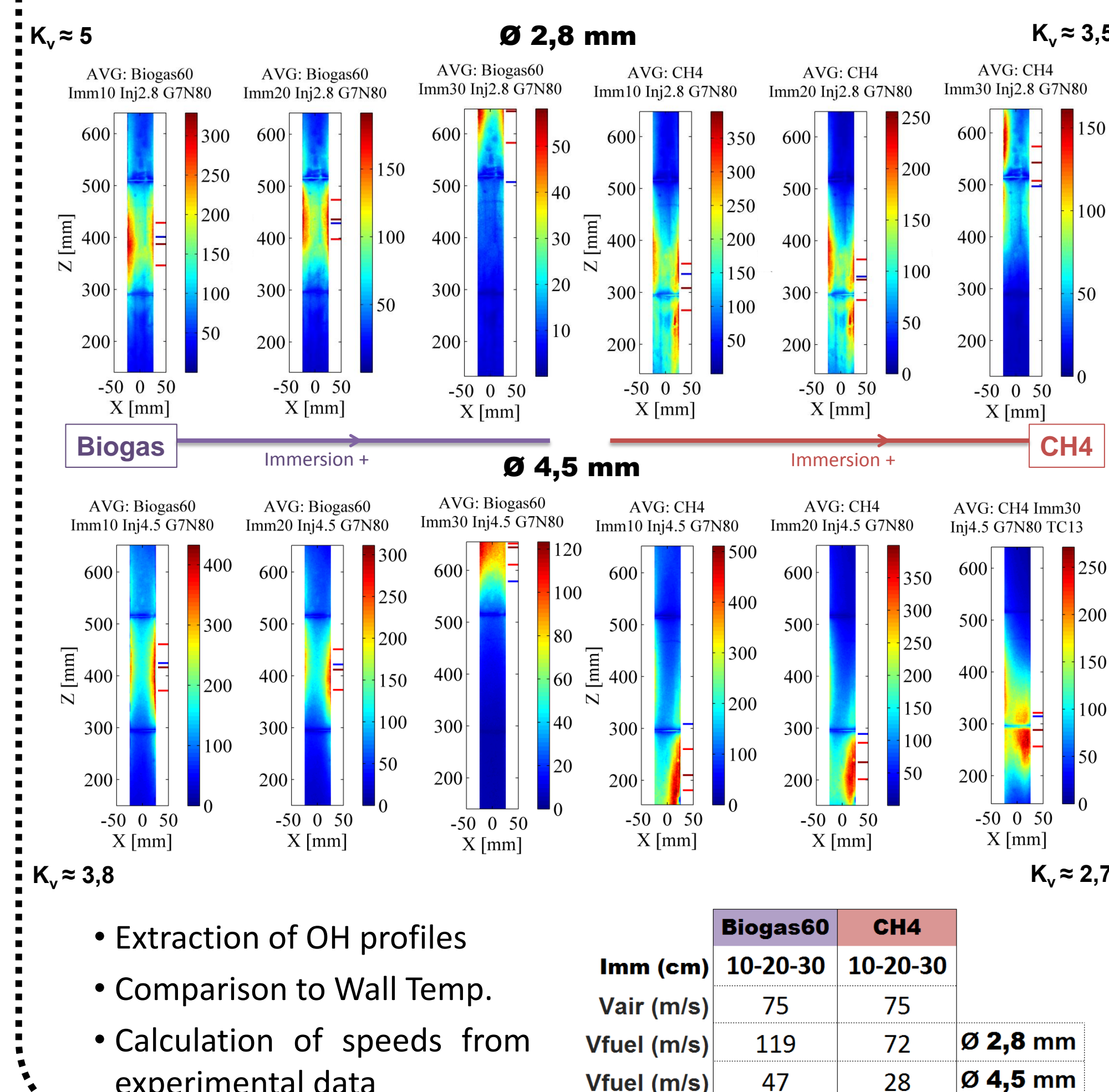
Heat balance

$$P_{\text{comb}} + S_{\text{hFuel}} + P_{\text{Air}} = P_{\text{Flue}} + P_{\text{Load}} + P_{\text{Wall}} + \Delta P$$

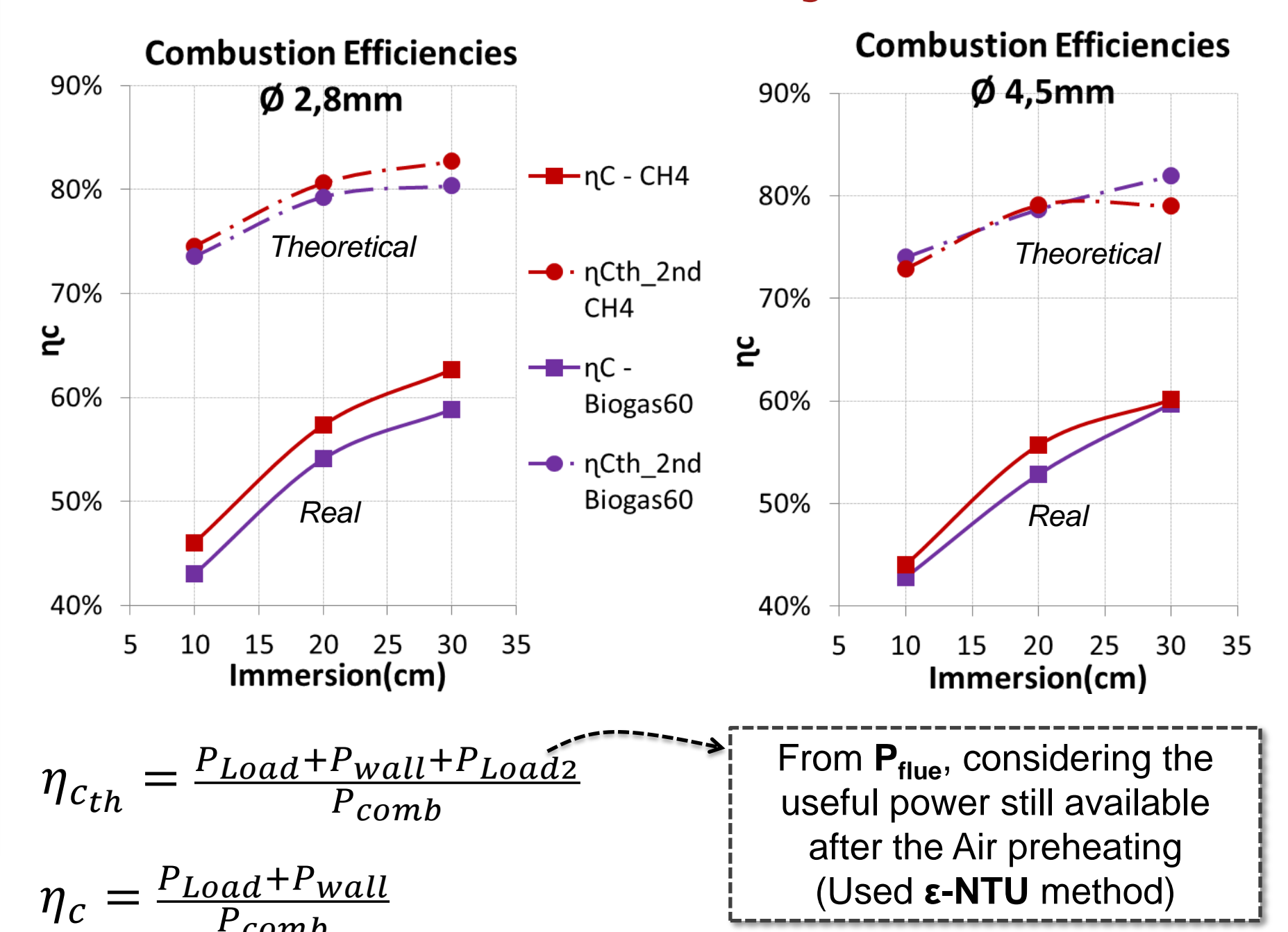
30 kW negligible P_{Losses}

	Biogas60			CH ₄			
Imm (cm)	10	20	30	10	20	30	
P _{flue} (kW)	20,3	18,7	17,6	19,6	18,0	17,1	Fuel Injectors Ø 2,8 mm
P _{load} (kW)	9,1	12,9	14,9	10,1	14,0	16,3	
P _{load} +P _{wall} (kW)	12,9	16,2	17,6	13,8	17,2	18,8	
ΔP (kW)	6,7	5,0	4,7	6,5	4,7	4,1	
P _{flue} (kW)	20,5	18,9	17,8	19,7	18,1	16,8	Fuel Injectors Ø 4,5 mm
P _{load} (kW)	9,4	13,0	15,5	9,9	13,8	16,0	
P _{load} +P _{wall} (kW)	12,8	15,8	17,9	13,2	16,7	18,0	
ΔP (kW)	6,5	5,1	4,2	7,0	5,1	5,1	

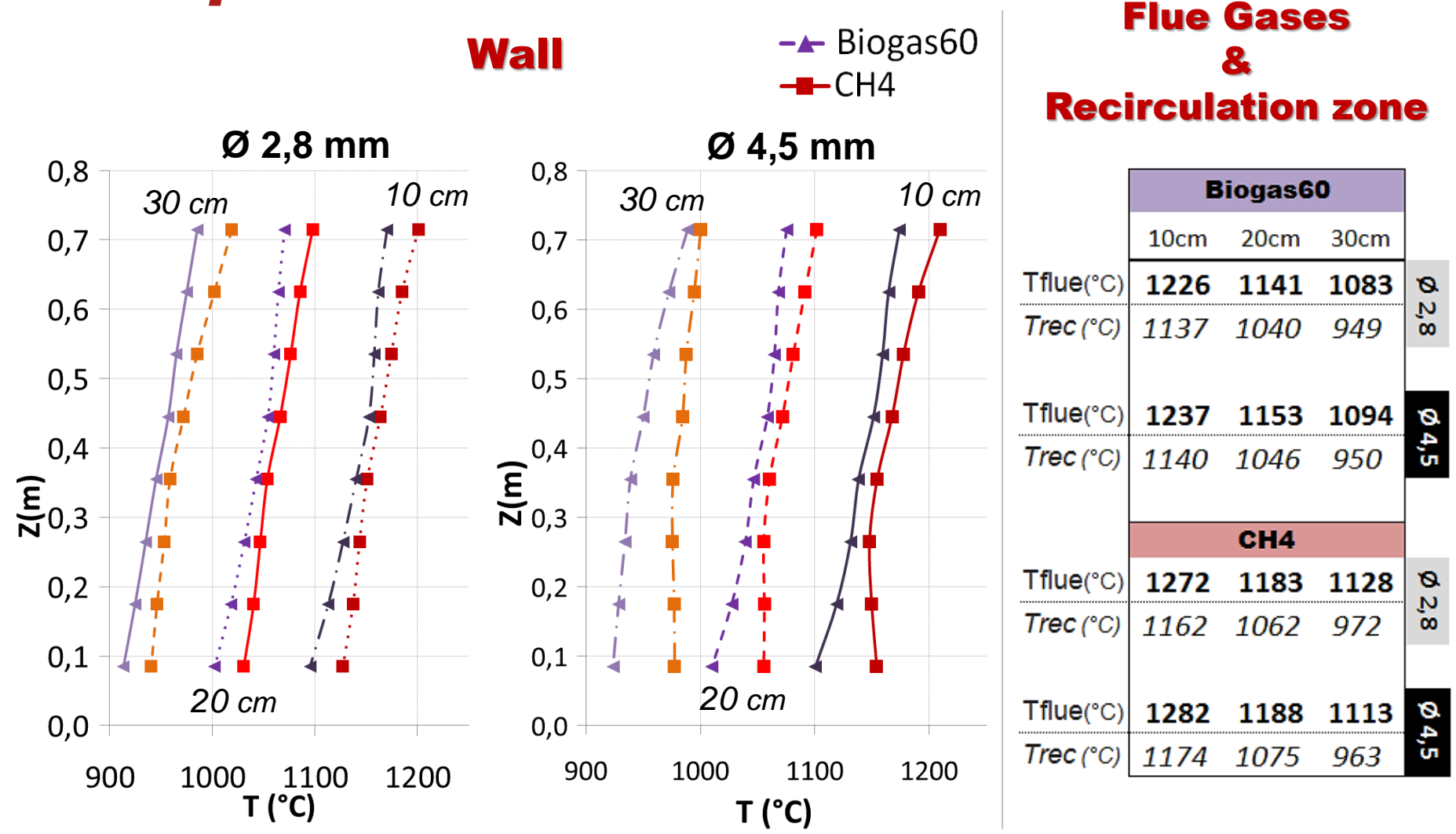
Images: OH



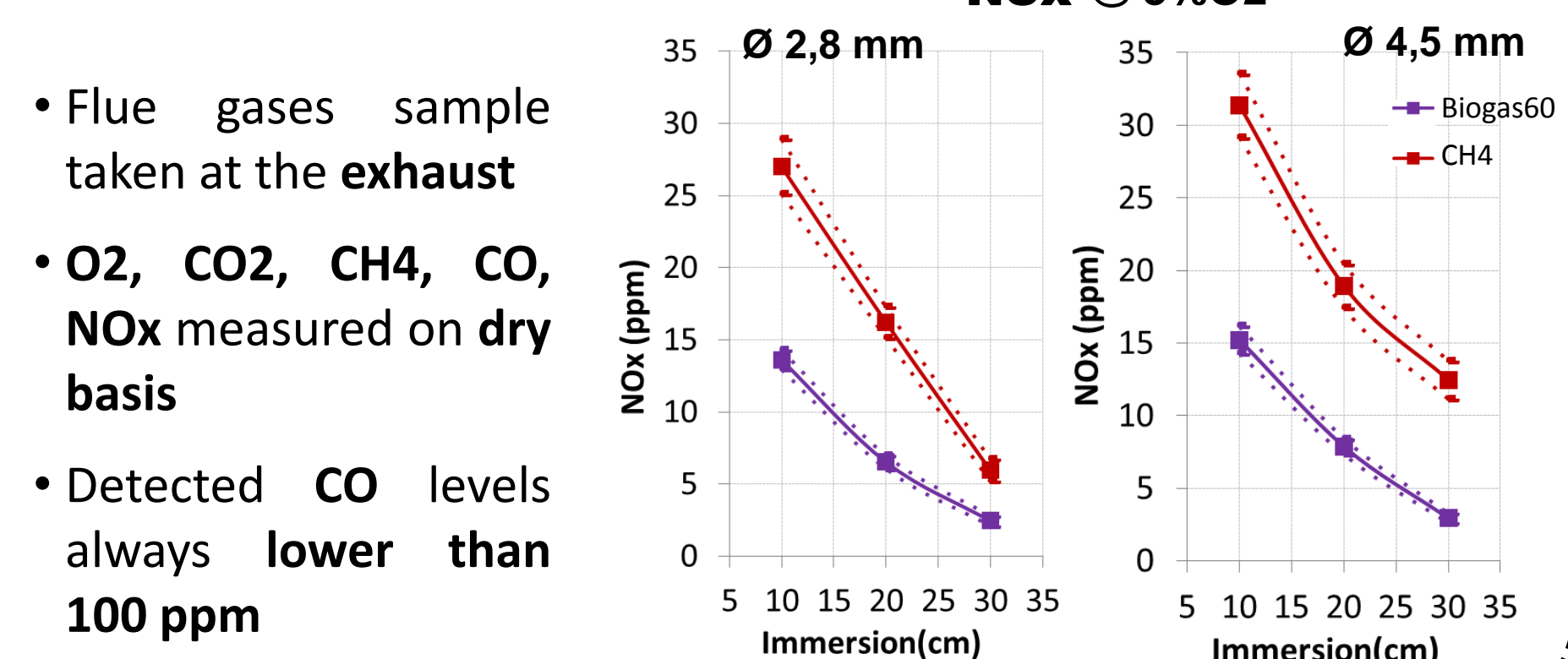
Combustion efficiency



Temperatures



Emissions



Main conclusions

- **Similar behavior** between Biogas and CH₄
- **No strong differences** in terms of efficiencies
- **Reduced wall temperatures & NO_x levels for Biogas** (cooling effect of CO₂ higher content)
- Higher losses when main reaction zone is placed at the bottom of the chamber
- **Biogas more sensitive** to the increase of the Load immersion, so to the decrease of flue gases temperature.
- Effects of **decreasing fuel injectors diameter**:
 - main reaction zones are more **stable** and do not show asymmetries;
 - the reaction zones are **more extended** along the chamber height due to the high momentum rate of fuel jets.